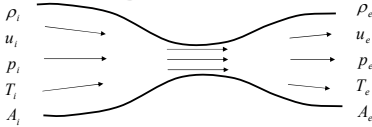


Variable Area Internal Flow

- Let's return now to internal flow, but now with area as a variable rather than constant.
- As shown below, this type flow is not 1-D anymore due to the curved wall surfaces and the streamlines along them.



- As problematic is the fact that the flow properties are not constant across the duct.

Variable Area Internal Flow [2]

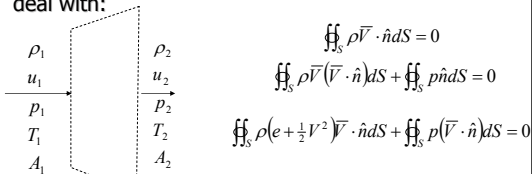
- However, a good approximation can be made by assuming uniform properties and horizontal flow – the Quasi 1-D Flow assumption.
- This is the same approximation used in fluid mechanics to obtain the pipe continuity equation:

$$\rho VA = \text{constant}$$

- In practice the longer the duct and thus the less curved the walls, the better the approximation.
- Short ducts with rapid area changes will not be well modeled – but those are usually not good flow situations anyway.

Quasi 1-D Flow Equations

- If a segment of the duct is used with known initial conditions, we can use our conservation equations to develop relations for the outflow conditions.
- The problem is very similar to the one for 1-D flow, but there are now two areas – both known – to deal with:



$$\iint_S \rho \vec{V} \cdot \hat{n} dS = 0$$

$$\iint_S \rho \vec{V} (\vec{V} \cdot \hat{n}) dS + \iint_S p \hat{n} dS = 0$$

$$\iint_S \rho \left(e + \frac{1}{2} V^2 \right) \vec{V} \cdot \hat{n} dS + \iint_S p (\vec{V} \cdot \hat{n}) dS = 0$$

Quasi 1-D Flow Equations [2]

- Applying the integral relations to the inflow and outflows (no flow through the sides) gives the familiar mass continuity equation:

$$\iint_S \rho \vec{V} \cdot \hat{n} dS = 0$$

$$-\rho_1 u_1 A_1 + \rho_2 u_2 A_2 = 0$$

$$\boxed{\rho_1 u_1 A_1 = \rho_2 u_2 A_2}$$

- A complication exists in applying the momentum equation because while there is not flow through the sides, there is a variable pressure there.

Quasi 1-D Flow Equations [3]

- Without yet knowing how pressure varies with area, that term must be left as an integral:

$$\iint_S \rho \vec{V} (\vec{V} \cdot \hat{n}) dS + \iint_{sides} p \hat{n} dS = 0$$

$$-\rho_1 A_1 - \rho_1 u_1^2 A_1 + p_2 A_2 + \rho_2 u_2^2 A_2 + \iint_{sides} p (\hat{i} \cdot \hat{n}) dS = 0$$

- Note that the sign of the dot product can be determined. So for an increasing area:

$$(\hat{i} \cdot \hat{n}) dS = -dA$$

- Thus:

$$\boxed{p_1 A_1 + \rho_1 u_1^2 A_1 + \iint_{A_1}^{A_2} p dA = p_2 A_2 + \rho_2 u_2^2 A_2}$$

Quasi 1-D Flow Equations [4]

- Finally, the energy conservation equation gives the same result as before:

$$\iint_S \rho \left(e + \frac{1}{2} V^2 \right) \vec{V} \cdot \hat{n} dS + \iint_S p (\vec{V} \cdot \hat{n}) dS = 0$$

$$-\rho_1 \left(e_1 + \frac{1}{2} u_1^2 \right) u_1 A_1 - p_1 u_1 A_1 + \rho_2 \left(e_2 + \frac{1}{2} u_2^2 \right) u_2 A_2 + p_2 u_2 A_2 = 0$$

$$-\rho_1 u_1 A_1 \left(e_1 + \frac{p_1}{\rho_1} + \frac{1}{2} u_1^2 \right) + \rho_2 u_2 A_2 \left(e_2 + \frac{p_2}{\rho_2} + \frac{1}{2} u_2^2 \right) = 0$$

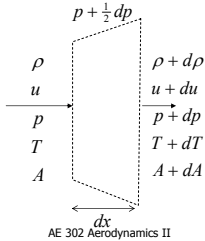
$$e_1 + \frac{p_1}{\rho_1} + \frac{1}{2} u_1^2 = e_2 + \frac{p_2}{\rho_2} + \frac{1}{2} u_2^2$$

$$\boxed{h_1 + \frac{1}{2} u_1^2 = h_2 + \frac{1}{2} u_2^2}$$

- Note that by our Quasi 1-D assumption, there is not normal velocity component – thus $u = V$!

Differential Equation Forms

- Because of the remaining integral term, the previous set of equation cannot be solved yet.
- Instead, choose a new, very thin control volume across which the properties change infinitesimally:



- We will also assume that the pressures on the sides can be represented adequately by the average of the initial and final pressures.
- This assumption is consistent with a linear approximation of the changes.

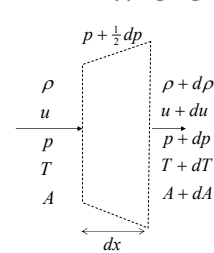
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Differential Equation Forms

- Mass conservation for this new control volume, after dropping higher order terms, becomes:



$$\iint_S \rho \vec{V} \cdot \hat{n} dS = 0$$

$$-\rho u A + (\rho + d\rho)(u + du)(A + dA) = 0$$

$$\boxed{\rho u dA + \rho A du + u A d\rho = 0}$$

- Or, dividing through by our previous continuity equation, $\rho u A = \text{constant}$:

$$\boxed{\frac{dA}{A} + \frac{du}{u} + \frac{d\rho}{\rho} = 0}$$

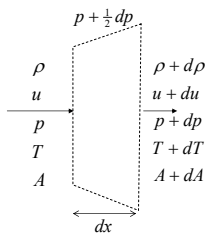
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Differential Equation Forms [2]

- Similarly for the momentum equation, expanding and dropping higher order terms gives:



$$\iint_S \rho \vec{V} (\vec{V} \cdot \hat{n}) dS + \iint_S p \hat{n} dS = 0$$

$$-\rho A - \rho u^2 A + (p + dp)(A + dA) + (\rho + d\rho)(u + du)^2 (A + dA) - (p + \frac{1}{2} dp) dA = 0$$

$$\boxed{A dp + \rho u^2 dA + 2\rho u A du + u^2 A d\rho = 0}$$

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Differential Equation Forms [3]

- Note that the Euler momentum equation can be obtained by subtracting the first form of continuity times velocity from this momentum equation:

$$Adp + \rho u^2 dA + 2\rho u A du + u^2 A d\rho - u(\rho u dA + \rho A du + u A d\rho) = 0$$

$$\boxed{dp + \rho u du = 0}$$

- However, a more interesting result is obtained by using the second form of continuity with the definition of the speed of sound:

$$d\rho = \left(\frac{\partial \rho}{\partial p}\right)_s dp = \frac{dp}{a^2}$$

Differential Equation Forms [4]

- Replacing the density derivative in continuity gives:

$$\frac{dA}{A} + \frac{du}{u} + \frac{dp}{\rho a^2} = \frac{dA}{A} + \frac{du}{u} - \frac{u du}{a^2} = 0$$

- Or, using the definition of Mach number:

$$\boxed{\frac{dA}{A} = (M^2 - 1) \frac{du}{u}}$$

- You might note that this equation, for incompressible flow ($M=0$) gives the expected inverse relation between velocity and area:

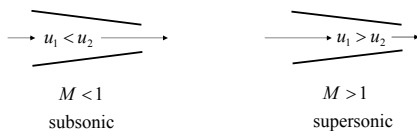
$$\boxed{\frac{dA}{A} = -\frac{du}{u}}$$

Velocity-Area Relations

- The really interesting thing about this equation is what it says about the difference between subsonic and supersonic compressible flow.

$$\boxed{\frac{dA}{A} = (M^2 - 1) \frac{du}{u}}$$

- If $M < 1$, then as area decreases, velocity increases. But if $M > 1$, the opposite occurs:

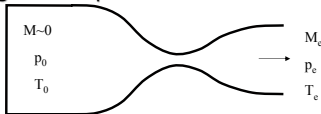


Nozzle Flows

- Up to now, the converging-diverging ducts have all had a sonic throat and supersonic exit.
- However, we now that that is not necessarily the case all the time – otherwise you could generate supersonic flow by squeezing a straw and blowing through it.
- What is missing is the impact of the upstream total pressure and the downstream or exit pressure.
- The combination of these two pressures will determine both if the flow ever goes supersonic – and if it is still supersonic at the exit.

Nozzle Flows [2]

- To see this, consider the special case of a nozzle between a large reservoir of high pressure air and exiting variable pressure chamber.

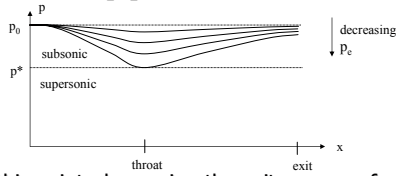


- We'll assume the reservoir is large enough that the velocity is near zero and the pressure remains constant during the time of interest.
- Now, consider what happens for different exit pressures.

Nozzle Flows [3]

- If the exit pressure is equal to the reservoir pressure... nothing happens. There will only be flow if there is a pressure difference.
- If the exit pressure is slightly less than the reservoir pressure, the flow begins to move but only slowly.
- The lower the exit pressure, the higher the velocity up until the velocity at the throat goes sonic.
- We can see this effect by plotting the local pressure versus location in the nozzle.

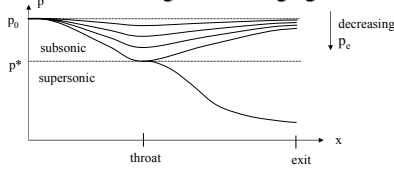
Nozzle Flows [4]



- At this point, decreasing the exit pressure further will not change the converging section any further since it is going as fast as it can at the throat.
- This situation is called having choked flow.
- Note that not even increasing the reservoir pressure would cause an increase in the velocities.

Nozzle Flows [5]

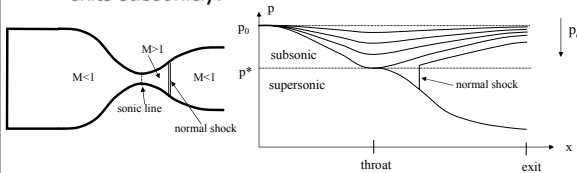
- Now consider the other extreme condition where the exit pressure is low enough to induce supersonic flow through the diverging duct.



- This flow condition is called fully-expanded.
- But what happens if the exit pressure is some where between these two extremes?

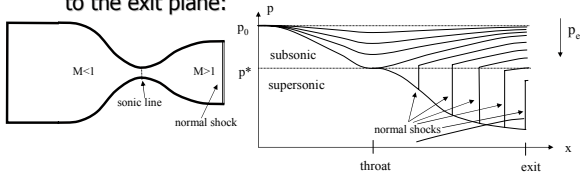
Nozzle Flows [6]

- For pressures below the initial choked condition, the flow goes supersonic beyond the throat.
- However, in order to exit with the correct pressure, the flow then passes through a normal shock, and exits subsonically.



Nozzle Flows [7]

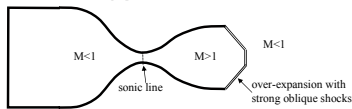
- With decreasing pressure, the shock moves further and further downstream the nozzle – until it gets to the exit plane:



- Note that due to losses in the shock, the exit pressure can be below the upstream p^* but the flow is subsonic – i.e. p^* decreases across the shock.

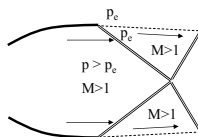
Over Expansion

- If the pressure is dropped further, we get a condition called over expanded flow.
- To get the correct outflow pressure, oblique shocks form on the nozzle lip.
- Initially, these are oblique shocks of the strong type – i.e. subsonic flow behind them – that turn normal where they join.



Over Expansion [2]

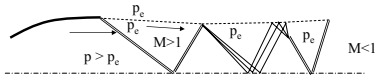
- As the pressure drops further, they become weak oblique waves with supersonic post-shock flow.
- However, since the flow is supersonic, the waves interact at the centerline and reflect as shocks:



- But what happens when the reflected shocks intersect the boundary between the nozzle flow and the ambient air?

Over Expansion [3]

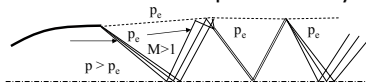
- This is a case we did not discuss earlier – a wave reflecting off of a pressure boundary.
- The rule here is that the wave reflects as a wave of the opposite family and opposite type.
- The change in type is required to ensure equal pressures across the pressure boundary.



- This diamond pattern repeats itself until the flow finally goes subsonic behind one shock.

Under Expansion

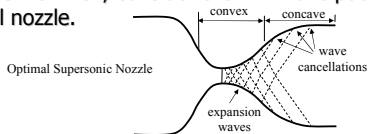
- The opposite situation – under expansion – occurs if the exit pressure is below the fully expanded value.
- In this case, the first set of waves are expansion waves – but otherwise the pattern is very similar.



- For optimum performance, there should be no waves – i.e. not over- or under-expanded but fully expanded flow.

Optimal Nozzle Design

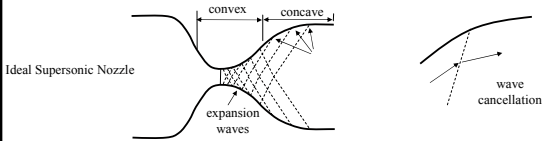
- So far, all the analysis performed for supersonic nozzles has relied upon Quasi 1-D assumptions.
- Before we finish, consider the 2-D wave pattern in a real nozzle.



- In the forward nozzle, the convex surface produces expansion waves which accelerate the flow.
- Most nozzles walls turn concave towards the exit to direct the flow into the thrust direction.

Ideal Nozzle Design [2]

- Normally, there would be compression waves in the concave wall region.
- However, in an optimal design, the expansion waves reflected back to the wall can be used to cancel out the compression waves.

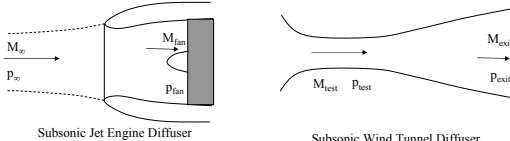


Diffusers

- In the previous section, we concentrated mainly on nozzles: ducts designed to accelerate a flow.
- Let's also consider diffusers: ducts designed to decelerate a flow.
- Actually, a better description of a diffuser is as a duct designed to achieve a pressure recovery – usually by slowing the flow.
- Since the pressure recovery is important, then the ratio of final to initial total pressure is used as the measure of diffuser efficiency.

Diffusers [2]

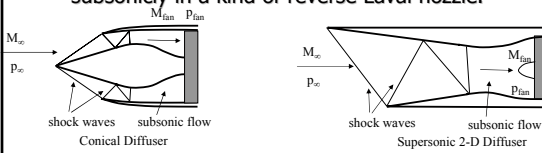
- If the flow of interest is subsonic, then diffusers are simply expanding area ducts as in the jet inlet or wind tunnel exit shown below.



- In both cases, the final velocity is slower and the final pressure higher than initially.
- The limit of how fast the duct area increases is due to the viscous flow on the walls.

Inlet Diffusers

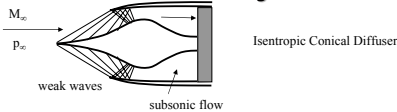
- However, if a flow is initially supersonic, then the flow must first be slowed to Mach 1.0 and then subsonically in a kind of reverse Laval nozzle.



- The most common design is a series of oblique shocks like the two inlet diffusers shown above.
- To make these designs work for a variety of flight speeds, movable surfaces need to be used.

Inlet Diffusers [2]

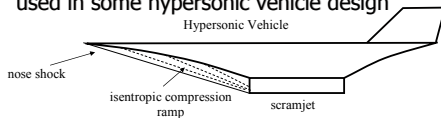
- Of course, a normal shock can be used like in a ramjet, but with a large loss in efficiency.
- Or you might consider a smooth concave ramp like that below to avoid shocks all together.



- However, this design must be longer and thus have a boundary layer whose total pressure loss is comparable to that of oblique shocks.
- Also this is hard to optimize for different speeds.

Inlet Diffusers [3]

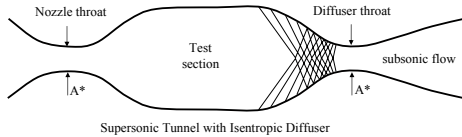
- However, isentropic compression ramps have been used in some hypersonic vehicle design



- At these speeds, the total pressure loss in shock waves is so high so as to make the trade against the boundary layer losses acceptable.

Wind Tunnel Diffusers

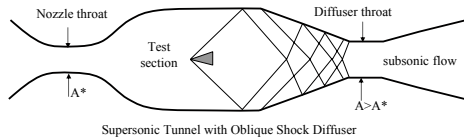
- Next consider the diffuser design in a supersonic wind tunnel.
- One possibly design is to use a smooth isentropic like that shown below.



- In this case there is a downstream throat with the same cross sectional area as the upstream one.

Wind Tunnel Diffusers [2]

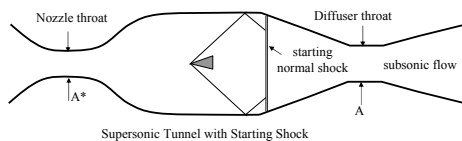
- However, with an unknown disturbance in the section, and for a wider range of operating conditions, an oblique shock type diffuser is usually used.



- Also, note that due to losses in the shocks, the downstream throat will have to be slightly larger than the upstream throat.

Wind Tunnel Diffusers [3]

- Another issue with wind tunnel diffuser design is the problem of initially starting the flow.
- It is well known that in the initial start up, after the upstream nozzle goes supersonic, a normal shock passes downstream through the test section to the diffuser.



Wind Tunnel Diffusers [4]

- This normal shock must be “swallowed” into the diffuser throat to complete the start up process.
- However, due to its greater total pressure loss, the throat area required to swallow the normal shock is greater than the area for normal operation.
- Thus, the tunnel must have a variable area diffuser, or compromise between optimum run performance and start-up capability.
- Note that early jet engine designs were notorious for inlet un-starting, i.e. un-swallowing a normal shock due to flow back-up through the engine.
